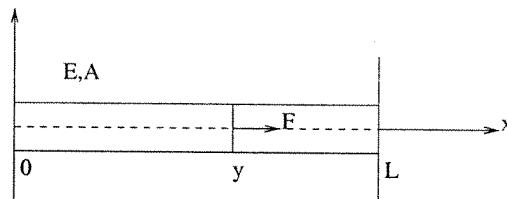


Exercise 1**Problem 1**

If you do not remember (or have not seen) the result $N(B) = R(B^T)^\perp$ from linear algebra, prove it (or look it up in your textbook).

Problem 2

Consider a rod (E, A, L) fixed at both ends and loaded by a force F at point y , ($0 < y < L$):



Find the expression for the displacement $u(x)$ in the form

$$u(x) = K(x, y)F.$$

Problem 3 (home exercise)

In the lectures we learned that the equilibrium equations

$$A^T C A x = f$$

are obtained from the principle of minimum potential energy. Show that an equivalent formulation is the principle of minimum complementary energy:

$$\min_y \frac{1}{2} y^T C^{-1} y \quad \text{subject to } A^T y = f.$$

Hint: Look up the technique of Lagrange multipliers.

Problem 4

Read through section 4. Dynamical systems and eigenvalues from the handouts.

Problem 5

A project work for the next two weeks. Hints will be given in exercises at week 4 and this will be **home exercise for week 5**.

Make your own MATLAB program by which simple strusses can be analyzed. Make experiments on systems with and without unique solutions. In the latter case, check the nullspaces of the equilibrium equation and the force compatibility conditions. Compute and print the eigenmodes of some simple structures.

Problem 1

$$N(B) = R(B^T)^\perp$$

$B: X \rightarrow Y$ lin. operator.

X, Y Banach spaces.

$$N(B) = \{x \in X \mid x \neq 0, Bx = 0\}$$

$$R(B^T) = \{x \in X \mid \exists y \in Y \text{ s.t. } B^T y = x\}$$

$$R(B^T)^\perp = \{x \in X \mid \exists y \in Y \text{ s.t. } B^T y = x$$

$$\text{and } \langle B^T y, z \rangle = 0 \quad \forall z \in R(B^T)$$

Take $x \in R(B^T)^\perp$.

$$\Leftrightarrow \langle x, z \rangle_x = 0 \quad \forall z \in R(B^T)$$

$$\Leftrightarrow \langle x, B^T y \rangle_x = 0 \quad y \text{ is arbitrary}$$

$$\Leftrightarrow \langle Bx, y \rangle_Y = 0 \quad \forall y \in Y$$

$$\Leftrightarrow Bx = 0$$

$$\Leftrightarrow x \in N(B)$$

Problem 2

Strain ϵ is uniform on $0 \leq x \leq y$ and $y < x \leq L$. Moreover $\epsilon = u'(x)$.

Therefore $u(x)$ is linear on both intervals.

$$u(x) = \begin{cases} a_1 x + b, & 0 \leq x \leq y \\ a_2 x + b_2, & y < x \leq L \end{cases}$$

Boundary conditions $u(0) = u(L) = 0$ imply

$$u(x) = \begin{cases} a_1 x & 0 \leq x < y \\ a_2(x-L) & y < x \leq L \end{cases}$$

We need two more conditions to deduce a_1 and a_2 . These are compatibility conditions

1) $u(x)$ is continuous i.e. $u(y_-) = u(y_+)$.

2) force compatibility

$$-A\sigma(y_-) - A\sigma(y_+) + f = 0$$

Continuity: $u(y_-) = u(y_+)$

$$a_1 y = a_2(y-L) \Leftrightarrow a_2 = \frac{y}{y-L} a_1$$

$$\Rightarrow u(x) = \begin{cases} a_1 x & 0 \leq x < y \\ a_1 \frac{y}{y-L} (x-L) & y < x \leq L \end{cases}$$

Force compatibility:

$$\sigma = E\varepsilon = Eu(x) = \begin{cases} Ea_1 & 0 \leq x < y \\ Ea_1 + \frac{y}{y-L} & y < x \leq L \end{cases}$$

$$\Rightarrow -A\sigma(y-) - A\sigma(y+) + f = 0$$

$$\Leftrightarrow AEa_1 \left(1 + \frac{y}{y-L}\right) = F$$

$$\Leftrightarrow a_1 = \frac{y-L}{AE(2y-L)} F$$

$$\Rightarrow u(x) = \begin{cases} \frac{y-L}{AE(2y-L)} Fx & 0 \leq x < y \\ \frac{y}{AE(2y-L)} F(x-L) & y < x \leq L \end{cases}$$

Define

$$K(x, y) := \begin{cases} \frac{y-L}{AE(2y-L)} x & 0 \leq x \leq y \\ \frac{y(x-L)}{AE(2y-L)} & y < x \leq L \end{cases}$$

$$\Rightarrow u(x) = K(x, y) F$$

Problem 3

$$\min_y \frac{1}{2} y^T C^{-1} y \text{ subject to } A^T y = f.$$

With Lagrange multipliers, the solution is at an extremum of

$$L(y, \lambda) = \frac{1}{2} y^T C^{-1} y + \lambda^T [f - A^T y]$$

$$\nabla_\lambda L(y, \lambda) = f^T - y^T A = 0 \quad (\text{Nothing new...})$$

$$\nabla_y L(y, \lambda) = \frac{1}{2} [(C^{-1}y)^T + y^T C^{-1}] - \lambda^T A^T = 0$$

$$\Leftrightarrow y^T C^{-1} - \lambda^T A^T = 0$$

$$\Leftrightarrow y^T = \lambda^T A^T C$$

Substitute this to $L(y, \lambda)$.

$$\begin{aligned} \Rightarrow \frac{1}{2} \lambda^T A^T \underbrace{C C^{-1}}_{=I} C A \lambda + \lambda^T f - \lambda^T A^T C A \lambda \\ = -\frac{1}{2} \lambda^T \underbrace{A^T C A}_{\text{positive definite}} \lambda + \lambda^T f \end{aligned}$$

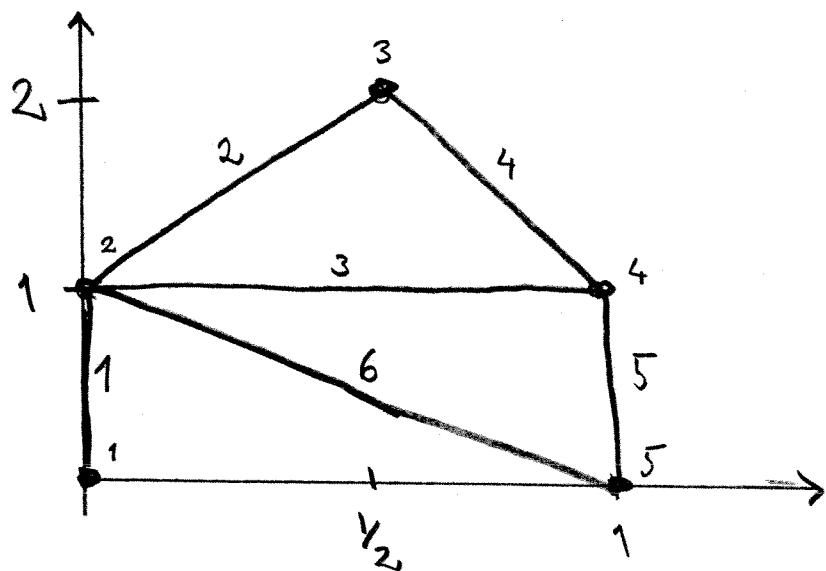
positive definite

\Rightarrow extremum is

$$\min_\lambda \frac{1}{2} \lambda^T A^T C A \lambda - \lambda^T f$$

Problem 5

Let us consider following truss



First we define joints

$$\text{joints} = \begin{bmatrix} 0 & 0 & \frac{1}{2} & 1 & 1 \\ 0 & 1 & 2 & 1 & 0 \end{bmatrix} \leftarrow \begin{array}{l} \text{x-coord} \\ \text{y-coord} \end{array}$$

Then we define bars

$$\text{bars} = \begin{bmatrix} 1 & 2 & 3 & 3 & 4 & 2 \\ 2 & 3 & 4 & 4 & 5 & 5 \end{bmatrix} \leftarrow \begin{array}{l} \text{1st joint} \\ \text{2nd joint} \end{array}$$

Now you can use `plot_truss.m` to draw the stress.

Next you need to loop through joints and build matrix A. See page 6 in handouts for help. Function `area2` could prove to be useful.

To get eigenmodes, you can use function eig (or eig).

Null spaces are perhaps easiest to check with singular value decomposition i.e. with function svd. Matlab svd gives you matrices U, S and V s.t.

$A = USV^T$, where S contains the singular values.